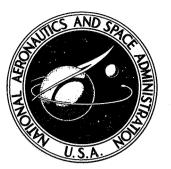
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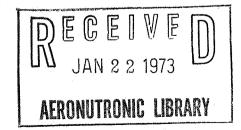
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HEAT-TRANSFER AND PRESSURE DISTRIBUTIONS ON HEMISPHERE-CYLINDERS IN METHANE-AIR COMBUSTION PRODUCTS AT MACH 7

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HEAT-TRANSFER AND PRESSURE DISTRIBUTIONS ON HEMISPHERE-CYLINDERS IN METHANE-AIR COMBUSTION PRODUCTS AT MACH 7

By Irving Weinstein Langley Research Center

SUMMARY

An experimental investigation was conducted to evaluate the use of a combustionproduct test medium for obtaining aerodynamic heating and loading data on large models. Three hemisphere-cylinder models were tested at angles of attack up to 18° in the Langley 8-foot high-temperature structures tunnel, which uses methane-air combustion products as the test medium. The tests were conducted at stagnation temperatures from 1400 K to 2056 K (2520° R to 3700° R) and free-stream unit Reynolds numbers from 1.3 × 10⁶ to 4.6 × 10⁶ per meter (0.4 × 10⁶ to 1.4 × 10⁶ per foot). The tunnel nominal free-stream Mach number was 7.0.

The results of tests at angles of attack from 0° to 18° showed that the longitudinal pressure distributions obtained over the surface of the models were in good agreement with data obtained in air and with theoretical predictions. Measured stagnation-point heating rates were within 7 percent of the theoretical value. Experimental heating distributions obtained at an angle of attack of 0° were also in good agreement with those obtained in air and predicted by theory.

INTRODUCTION

Aerodynamic test facilities that can simulate the heating and loading encountered in flight are indispensable tools for developing structural technology for supersonic and hypersonic vehicles. For effective structural testing, these facilities must accommodate large structural components and must provide a realistic aerothermal environment. The Langley 8-foot high-temperature structures tunnel is a large hypersonic wind tunnel which was designed specifically for such testing. This facility provides an aerothermal environment representative of Mach 7 flight by burning methane under high pressure in air and using the resulting combustion products as a test medium.

Although the use of combustion products as a test medium was previously explored in a small combustion facility (ref. 1), it was necessary to evaluate the test stream of the present facility to establish its suitability for aerothermally testing large models. Accordingly, aerodynamic heating and pressure distributions were measured on three hemisphere-cylinder models up to 61 cm (24 in.) in diameter. The models were tested at angles of attack up to 18° , at stagnation temperatures from 1400 K to 2056 K (2520° R to 3700° R), and at free-stream Reynolds numbers from 1.3×10^{6} to 4.6×10^{6} per meter (0.4×10^{6} to 1.4×10^{6} per foot). The measured distributions presented herein are compared with other distributions obtained in air and those predicted by theory.

SYMBOLS

The units used for physical quantities defined in this paper are given both in the International System of Units (SI) and in the U.S. Customary Units. The calculations were made in U.S. Customary Units. Factors relating the two systems are given in reference 2 and in the appendix.

с	specific heat of material, J/kg- ⁰ K (Btu/lb- ⁰ R)
М	Mach number
р	pressure, N/m^2 (lb/in ²)
q	dynamic pressure, N/m ² (lb/ft ²)
ģ	heating rate, W/m^2 (Btu/ft ² -sec)
R	free-stream Reynolds number based on model diameter
r	model nose radius, m (ft)
S	distance along model surface measured from stagnation point, m (ft)
ŝ	distance along model measured from axis of symmetry, m (ft)
Ť	temperature, K (^O R)
t	time, sec
α	angle of attack, deg
•	

β	meridian	angle,	deg

δ	standoff	distance	of	bow	shock	wave

 θ angle between model axis and inward normal to surface, deg

 ρ material density, g/m³ (lb/ft³)

 τ model skin thickness, m (ft)

Subscripts:

l	local	static	conditions

s model stagnation point

t total conditions

 ∞ free-stream conditions

MODELS

Three hemisphere-cylinder models were used for this investigation. One of the models was used for both heat-transfer and pressure measurements; a second model was used for pressure measurements only; and a third model was used for heat-transfer measurements only. These models are designated as models A, B, and C, respectively, and are described in subsequent sections.

Model A

A sketch of model A and a photograph of the model mounted in the test section of the Langley 8-foot high-temperature structures tunnel are shown in figure 1. The model, which was 30.48 cm (12.00 in.) in diameter and 0.9 m (36 in.) long, was constructed of 347 stainless steel with a wall thickness of 0.24 cm (0.094 in.) and a surface finish of 0.4 μ m (16 μ in.) root mean square.

The model was instrumented with twenty-four 0.152-cm-diameter (0.060 in.) surface pressure orifices (see table in fig. 1) which were arranged in a spiral to reduce possible interference effects. The orifices were connected to pressure transducers

located within the model cavity with short lengths, 0.3 to 1.2 m (1 to 4 ft), of stainlesssteel tubing to minimize the response time of the transducers. The model was also instrumented with 31 chromel-alumel thermocouples of No. 30 wire which were welded to the back surface of the model skin. All but two of these thermocouples were located in the vertical plane of symmetry.

Model B

A sketch of model B and a photograph of the model in the tunnel test section are shown in figure 2. The model, which was 61 cm (24 in.) in diameter and 91 cm (36 in.) long, was constructed of a mild carbon steel and had a wall thickness of 1.3 cm (0.5 in.) and a surface finish of 1.6 μ m (63 μ in.) root mean square.

The model was instrumented with eleven 0.152 cm (0.060 in.) inside diameter pressure orifices which were located as shown in the table of figure 2. The orifices were connected to pressure transducers which were located inside the model to reduce the tubing length; as a result, a more rapid pressure response occurs.

Model C

A sketch and a photograph of model C are shown in figure 3. This model, which was 35.7 cm (14.06 in.) in diameter and 57 cm (22.5 in.) long, was made of Inconel and had a wall thickness of 0.08 cm (0.031 in.) and a surface finish of 0.4 μ m (16 μ in.) root mean square.

The model was instrumented with 26 chromel-alumel thermocouples of No. 30 gage wire with each leg resistance welded separately to the back surface of the model skin. All but two of these thermocouples were located in the vertical plane of symmetry.

APPARATUS AND TEST PROCEDURES

Facility

The present investigation was conducted in the Langley 8-foot high-temperature structures tunnel. A schematic drawing of this facility is shown in figure 4. A brief description and operating conditions of this tunnel are reported in references 3 and 4. The facility is a hypersonic blowdown wind tunnel that burns methane in air under high pressure to produce the high energy gases required for simulation of the aerothermal flight environment. The resulting products of combustion are expanded through an axisymmetric contoured nozzle having an exit diameter of 2.4 m (8 ft). The nozzle provides essentially parallel flow at a nominal Mach number of 7 in an open jet test section. A single-stage annular air ejector located downstream of the test section is used to

establish supersonic flow and to permit operation at low stagnation pressures. The tunnel test section is 4.3 m (14 ft) long and has a usable test core approximately 1.2 m (4 ft) in diameter. Models are sting mounted on an elevator which inserts the model into the stream after test conditions are established. A model-pitching system provides for angles of attack from -20° to 20° .

The tunnel can be operated over a total temperature range from approximately 1389 K to 2111 K (2500° R to 3800° R). The dynamic-pressure range of the tunnel is from 14 kN/m² to 86 kN/m² (300 lb/ft^2 to 1800 lb/ft²) and the free-stream Reynolds number range is 1.1×10^6 to 9.8×10^6 per meter (0.35×10^6 to 3.0×10^6 per foot). These conditions simulate the hypersonic flight environment in the altitude range between 25 km and 40 km (80 000 ft and 130 000 ft).

The facility is equipped with a Z-shaped single-pass schlieren flow-visualization system which can be traversed longitudinally along the test section. The field of view is 0.6 m (2 ft) in diameter, and the mirrors are paraboloidal and have a focal length of 6 diameters. The high-intensity light source uses a xenon X-75 compact arc lamp which can be operated in either a continuous or flash mode.

Test Procedure

After equilibrium flow conditions were established in the tunnel, the models were elevated to the stream center line within 1 second to provide essentially a step-function exposure to the test conditions. The heat-transfer models (models A and C) were tested only at an angle of attack of 0° and were withdrawn from the stream after about 8 seconds to prevent overheating of the thin walls. The pressure distribution model B was inserted at 0° and then pitched to the desired angle where it was allowed to dwell for approximately 5 seconds to allow transducer outputs to stabilize before it was pitched to the next angle. The angle-of-attack range was from 0° to approximately 18° .

The test data were obtained over a model stagnation pressure range from 28 to 96 kN/m^2 (4 to 14 psia) at total temperatures from approximately 1400 K to 2056 K (2520° R to 3700° R). These conditions correspond to free-stream unit Reynolds numbers of 1.3×10^6 to 4.6×10^6 per meter (0.4×10^6 to 1.4×10^6 per foot).

Data Reduction

Outputs of thermocouples and pressure transducers were recorded on the Langley central data recording system at a sampling rate of 20 frames per second. Free-stream conditions in the test section were obtained from conditions measured in the combustor by using the results of previous test-section surveys.

The local convective heating rates were obtained by using the one-dimensional transient thin-wall technique which equates the heat entering the surface to the heat stored so that $\dot{q}_l = \rho c \tau \frac{dT}{dt}$. The temperature rise rate dT/dt was evaluated at a time when the model reached the center line of the test stream, which occurred at approximately the time of maximum temperature rise rate. At that time, the surface temperature was low, averaging about 344 K (620° R) at the stagnation point and 306 K (550° R) on the cylinder. Consequently, heat conduction through and along the wall and heat lost by radiation from the surface were considered to be negligible.

Pressure distributions were normalized by using measured stagnation point pressures. Since model A had no pressure orifice at the stagnation point, Newtonian theory was used to derive the stagnation pressure from the pressure measured at $\theta = 5^{\circ}$. The thermal, transport, and flow properties of methane-air combustion products of reference 5 were used in reducing all data and in computing theoretical values.

Accuracy

Accuracies of the parameters presented are estimated to be generally within the following limitations:

Mach number, M_{∞}	±0.1
Model stagnation pressure, p_s , percent	±2
Dynamic pressure, q, percent	±
Local convective heating rate, \dot{q} , percent	±6
Total temperature, T _t , K (^o R)	± 42 (±75)
Angle of attack, α , deg	±0.1
Shock standoff distance, δ , percent	±6

RESULTS AND DISCUSSION

Test conditions for all models and measured stagnation pressures and heating rates are presented in table I.

Pressure Distribution

Values of the nondimensional pressures obtained from model A are presented in table II and are plotted in figure 5. Figures 5(a) and 5(b) present the data obtained at the higher temperatures from 1833 K to 2056 K (3300° R to 3700° R), whereas figures 5(c) and 5(d) show the distributions obtained at the lower temperature range from 1400 K to 1472 K (2520° R to 2650° R). The theoretical curves shown were obtained with the aid of the computer program described in reference 6 which uses a blunt-body solution in the

region of subsonic and transonic local velocities, and the method of characteristics where the local velocities are supersonic. The program was modified to include the thermodynamic and transport properties of methane-air combustion gases; however, comparative calculations indicate that the use of combustion gas properties produces only about a 1-percent change in the distributions and in the absolute pressure level at the stagnation point. Variations of the test conditions for a given part of figure 5 produced less than a 2-percent change in the calculated distributions; consequently, a single representative theoretical curve is presented in each part of the figure. The theoretical and experimental distributions are in good agreement over the surface of the model for the lowtemperature cases of figures 5(c) and 5(d), the maximum variation of only 8 percent occurring at the aft end of the cylinder. The variation was greater for the higher temperature tests, the theory underpredicting the pressures by a maximum of 25 percent at the aft end of the cylinder (fig. 5(a)).

The pressure distribution range for the tests of model A are compared in figure 6 with data obtained on other hemisphere-cylinder models tested in air at Mach numbers from 4.63 to 8.10. The present data are in very good agreement with all the data obtained in air from references 7 and 8 over the surface of the hemisphere and are bracketed by the data of reference 8 at Mach numbers of 6.03 and 8.10 along the entire length of the cylinder.

Pressure distributions for model B at angles of attack up to about 18° are shown in figure 7. The data presented were obtained from orifices along the windward meridian of the model. The theoretical results were obtained by using the method of reference 6 by treating the hemisphere-cylinder as a spherically blunted cone with a half-angle equal to the model angle of attack. Although this simplified approach does not account for cross-flow effects, the theoretical and experimental distributions are in good agreement. Fairings of data along the cylinder obtained from reference 9 in air at a Mach number of 10.05 for angles of 0° , 10° , and 15° and presented in this figure are also in good agreement with the present data.

Heating Distributions

The experimental stagnation-point heating rates for model A, listed in table I, are compared with theoretical values in figure 8. The theoretical values were obtained from the viscous solution of reference 6 which is based on the stagnation-point solution of Fay and Riddell (ref. 10) and altered to include the properties of combustion products. The solid line represents the condition where theory equals experiment and the dashed lines show a ± 5 -percent variation with theory. The theory predicts the experimental data very well and is within 7 percent for all the present test data.

Nondimensionalized heating-rate distributions \dot{q}/\dot{q}_s for model A are listed in table III and are presented in figure 9. The theoretical curves on these plots are based on combustion products properties and were obtained with the aid of the computer program of reference 6 which used the laminar boundary-layer solution of Cohen (ref. 11) to obtain the heating rates over the model surface. A single theoretical curve represents all the data on each part of this figure. It can be seen that the experimental data and the theory for model A are in very good agreement over the entire surface of the model except at the aft end of the cylinder at the highest Reynolds number where the heating is underpredicted by laminar theory by approximately 19 percent. This variation may be the effect of transitional flow at the higher Reynolds number.

The heating distributions for model C are listed in table IV and presented in figure 10. The data show some scatter but are generally in good agreement with the theory of reference 6. A comparison of the nondimensional heating distributions for this model with those of model A at comparable test conditions indicates no apparent effect of model wall thickness.

A correlation of heating distributions obtained in combustion products with those obtained in air is shown in figure 11. Typical nondimensional heating distributions for models A and C are shown to be in reasonable agreement with combustion products data from reference 1 and with the theory of reference 6. These data show no significant effect of model diameter on the nondimensional heating distribution along the model surface even though the stagnation-point heating rate is higher for the model with the smaller radius. Therefore, if the heating at the stagnation point is known, the heating along the entire model surface can be predicted. Data obtained in air (from ref. 1) are shown as the solid symbols in figure 11. It can be seen that heating distributions obtained in a combustion-products test medium are in good agreement with those obtained in air and with those predicted by theory.

Flow Patterns

Typical schlieren photographs are shown in figure 12 for tests of model A at three different Reynolds numbers. No perceptible differences in the bow-shock shapes are indicated. The shock-standoff distances measured from enlargements of these photo-graphs are presented in figure 13 and are compared with other experimental data and with theory. The experimental data show that the shock-standoff distance was somewhat less in combustion products than in air but within the range predicted by the theories of references 12 to 14 for air. The standoff distances predicted by the theory of reference 6 using the properties of methane-air combustion products are in good agreement with the experimental data of the present test.

CONCLUDING REMARKS

Heat-transfer and pressure distributions were obtained on three hemispherecylinder models in a test medium consisting of methane-air combustion products in the Langley 8-foot high-temperature structures tunnel. Data were obtained at total temperatures from approximately 1400 K to 2056 K (2520° R to 3700° R) for free-stream Reynolds numbers from 1.3×10^{6} to 4.6×10^{6} per meter (0.4×10^{6} to 1.4×10^{6} per foot). The tests were made at a nominal Mach number of 7.0.

Pressure distributions over the surface of the model at an angle of attack of 0° and along the windward meridian at angles of attack up to 18° were generally in good agreement with data obtained in air and with theoretical predictions. For all cases the agreement between theory and experiment was very good over the hemisphere whereas the agreement along the cylinder was dependent upon the test conditions. The maximum variations, which occurred at the aft end of the cylinder, ranged from an underprediction by the theory of approximately 25 percent at higher temperatures to an overprediction of approximately 8 percent at the lower temperatures.

Measured stagnation-point heating rates were within 7 percent of the theoretical values. Heating distributions obtained at an angle of attack of 0° were in very good agreement with results obtained in air and with theoretical predictions except for a small region near the aft end of the cylinder. In this region the heating was underpredicted by a maximum of 19 percent and occurred at the higher values of Reynolds numbers of the tests. The flow condition was believed to be transitional.

Langley Research Center,

National Aeronautics and Space Administration, Hampton, Va., November 21, 1972.

APPENDIX

CONVERSION OF U.S. CUSTOMARY UNITS TO SI UNITS

Factors required for converting U.S. Customary Units to the International System of Units (SI) are given in the following table:

Physical quantity	U.S. Customary Unit	Conversion factor (*)	SI Unit (**)
Heat flux	Btu/ft 2 -sec	11 348.9	watts/meter ² (W/m^2)
Length	in. per ft	0.0254 3.28	meter (m) per meter (m ⁻¹)
Pressure	$\left\{\begin{array}{c} 1\mathrm{bf/in^2}\\ 1\mathrm{bf/ft^2} \end{array}\right.$	$6894.757 \\ 47.88026$	ight brace newtons/meter ² (N/m ²)
Temperature	oR	5/9	kelvin (K)

^{*}Multiply value in U.S. Customary Unit by conversion factor to obtain equivalent value in SI Unit.

**Prefixes to indicate multiples of units are as follows:

Prefix	Multiple
mega (M)	10 ⁶
kilo (k)	10 ³
centi (c)	10 ⁻²
micro (μ)	10-6

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TABLE I.- TEST CONDITIONS AND EXPERIMENTAL RESULTS

FOR HEMISPHERE-CYLINDER MODELS

			T _t		q	q _∞		p _s		ġ _s	
Run	α , deg	M_{∞}			R				-	/ 9	· · · · · · · · · · · · · · · · · · ·
			K	⁰ R		kN/m^2	lb/ft ²	kN/m^2	psia	MW/m ²	Btu/ft ² -sec
						Model	Α	<i></i>			
1	0	7.06	1933	3480	$0.55 imes 10^6$	24.7	516	46.47	6.74	0.369	32.52
2	0	7.36	2056	3700	.38	17.9	373	33.09	4.80	.358	31.56
3	0	6.22	1461	2630	.68	22.4	467	43.09	6.25	.255	22.50
4	0	6.15	1400	2520	.51	15.6	326	29.92	4.34	.193	16.98
5	0	6.95	1922	3460	.84	37.4	78 2	69.98	10.15	.479	42.20
6	0	6.17	1456	2620	1.01	33.1	691	63.64	9.23	.309	27.24
7	0	7.31	1972	3550	.37	16.1	337	29.51	4.28	.316	27.81
8	0	7.13	1961	3530	.53	23.7	496	44.40	6.44	.382	33.62
9	0	6.85	1900	3420	1.13	50.8	1062	94.67	13.73	.567	50.00
10	0	6.14	1472	2650	1.38	46.3	966	88.80	12.88	.386	34.00
11	0	6.90	1833	3300	.56	23.3	486	43.99	6.38	.356	31.37
	•		•			Model	В	·····			
12	-0.2	6.82	1778	3200	$1.26 imes 10^6$	25.5	532	48.26	7.00		
12	17.8	6.82	1778	3200	1.26	25.5	532	48.26	7.00		
13	3	6.78	1756	3160	1.22	24.7	515	46.75	6.78		
	11.1										
	13.0				· · · · ·						
	14.9										
V	16.9	¥	₩	V	• ↓	V I	V	V	V		
			•	•	L	Model	С		L	 	L
14	0	7.07	1922	3460	0.63×10^{6}	23.3	487	*43.71	6.34	0.404	35.60
15	0		1644	2960		43.7	913	*83.57	12.12	.418	36.83

*Values determined theoretically.

θ,	β,	₅/r	<u>,</u>		• • • • • • • • •		p/p	for ru	n –			anganan di seta dan panan	.
deg	deg	5/ r	1	2	3	4	5	6	7	8	9	10	11
*5	75	0.087											
10	285	.175	0.972	0.976	0.970	0.970	0.963	0.966	0.973	0.970	0.964	0.965	0.969
20	135	.349	.851	.837	.865	.848	.858	.866	.843	.847	.867	.870	.860
30	165	.524	.739	.746	.754	.760	.733	.748	.752	.738	.730	.742	.751
40	195	.698	.574	.579	.584	.588	.566	.586	.586	.572	.565	.578	.581
45	225	.785	.484	.480	.493	.490	.477	.492	.488	.482	.489	.494	.491
50	255	.873	.399	.396	.398	.391	.395	.404	.385	.392	.399	.404	.394
60	105	1.047	.270	.269	.270	.269	.266	.268	.270	.270	.288	.269	.272
60	285	1.047	.260	.258	.257	.254	.286	.293	.273	.277	.295	.303	.303
70	315	1.222	.1632	.1636	.1636	.1621	.1620	.1620	.1648	.1638	.1894	.1628	.1655
80	345	1.396	.0953	.0973	.0951	.0952	.0936	.0922	.0979	.0959	.1028	.0919	.0973
85	15	1.484	.0715	.0504	.0566	.0708							
90	45	1.571	.0526	.0555	.0508	.0516	.0520	.0489	.0558	.0534	.0513	.0487	.0545
-	225	1.738	.0481	.0510	.0455	.0465	.0483	.0453	.0505	.0490	.0482	.0450	.0494
	75	2.071	.0380	.0461	.0420	.0428	.0445	.0406	.0459	.0448	.0447	.0406	.0457
1	285	2.238	.0352	.0443	.0414	.0420	.0434	.0408	.0441	.0439	.0434	.0405	.0444
	135	2.571		.0409	.0380	.0386	.0403	.0369	.0407	.0400	.0406	.0370	.0413
	45	2.904	.0380	.0400	.0358	.0372	.0378	.0350	.0396	.0384	.0378	.0346	.0392
	225	2.904		.0408	.0380	.0385	.0403	.0366	.0410	.0405	.0404	.0366	.0412
1	195	3.571	.0333	.0362	.0326	.0341	.0340	.0314	.0358	.0346	.0338	.0309	.0356
	165	4.238	.0301	.0320	.0292	.0309	.0303	.0280	.0320	.0308	.0302	.0275	.0318
	315	4.738	.0301	.0319	.0286	.0303	.0292	.0271	.0317	.0303	.0293	.0265	.0311
	90	5.071	.0289	.0312	.0275	.0293	.0281	.0260	.0309	.0292	.0280	.0254	.0302
	270	5.571	.0276	.0302	.0263	.0282	.0268	.0248	.0299	.0280	.0265	.0241	.0289

TABLE II. - PRESSURE DISTRIBUTIONS p/p_s on model A

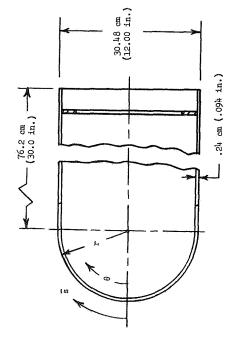
 $*5^{0}$ location used to obtain Newtonian stagnation values which were then used to nondimensionalize all pressures.

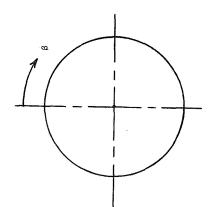
	I		\dot{g}/\dot{g}_{c} for run –											
θ,	β,	$\bar{\mathbf{s}}/\mathbf{r}$				·	ġ/ġs	for rur						
deg	deg		1	2	3	4	5	6	7	8	9	10	11	
0	0	0	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000	
5	0	.087	.981	.985	.982	.981	.977	.964	.983	.984	.972	.957	.982	
10	.0	.175	.960	.972	.954	.966	.960	.943	.973	.973	.953	.930	.965	
10	180	.175	.951	.957	.942	.906	.945	.931	.968	.958	.993	.925	.966	
20	0	.349	.889	.898	.881	.885	.891	.861	.902	.900	.883	.850	.890	
30	0	.524	.775	.784	.700	.781	.780	.753	.786	.785	.773	.742	.780	
40	0	.698	.637	.637	.630	.639	.647	.618	.643	.644	.637	.608	.638	
40	90	.698	.637	.634	.626	.639	.639	.591	.638	.644	.632	.608	.635	
40	180	.698	.610	.616	.609	.621	.615	.600	.619	.613	.593	.592	.544	
40	270	.698	.625	.625	.623	.623	.645	.613	.607	.632	.634	.605	.639	
45	0	.785	.566	.572	.565	.567	.572	.550	.567	.568	.569	.535	.562	
50	0	.873	.496	.494	.494	.491	.500	.477	.496	.499	.499	.471	.497	
60	0	1.047	.354	.348	.354	.342	.354	.338	.355	.357	.355	.332	.354	
70	0	1.222	.247	.233	.242	.239	.242	.229	.240	.244	.241	.231	.242	
80	0	1.396	.1545	.1456	.1559	.1721	.1522	.1494	.1545	.1552	.1564	.1471	.1553	
80	180	1.396	.1442	.1390	.1399	.1320	.1498	.1360	.1475	.1598	.1492	.1346	.1459	
85	0	1.484	.1182	.1159	.1191	.1205	.1218	.1140	.1189	.1206	.1232	.1146	.1196	
90	0	1.571	.0898	.0871	.0879	.0881	.0903	.0828	.0971	.0932	.0928	.0865	.0892	
	0	1.738	.0757	.0684	.0710	.0645	.0768	.0692	.0746	.0751	.0763	.0687	.0737	
	0	2.029	.0691	.0603	.0622	.0625	.0680	.0593	.0667	.0674	.0685	.0617	.0661	
	-0	2.238	.0621	.0580	.0606	.0554	.0629	.0528	.0613	.0623	.0633	.0566	.0606	
	180	2.238	.0526	.0591	.0542	.0557	.0596	.0493	.0546	.0585	.0552	.0534	.0579	
	0	2.571	.0538	.0520	.0509	.0488	.0560	.0483	.0543	.0551	.0560	.0516	.0551	
	0	2.904	.0506	.0463	.0482	.0440	.0497	.0434	.0460	.0502	.0496	.0461	.0461	
	0	3.238	.0434	.0405	.0439	.0412	.0457	.0416	.0460	.0473	.0440	.0425	.0469	
	0	3.571	.0401	.0393	.0403	.0407	.0418	.0376	.0429	.0418	.0425	.0392	.0418	
ł	0	3.904	.0385	.0378	.0360	.0351	.0395	.0368	.0369	.0397	.0386	.0399	.0410	
	180	3.904	.0362	.0375	.0350	.0348	.0414	.0352	.0380	.0373	.0374	.0347	.0392	
	0	4.571	.0344	.0346	.0354	.0341	.0337	.0319	.0338	.0351	.0352	.0330	.0384	
	0	5.071	.0341	.0297	.0314	.0316	.0317	.0322	.0324	.0319	.0324	.0338	.0342	
	0	5,571	.0288	.0282	.0314		.0318		.0298	.0290	.0302	.0350	.0303	

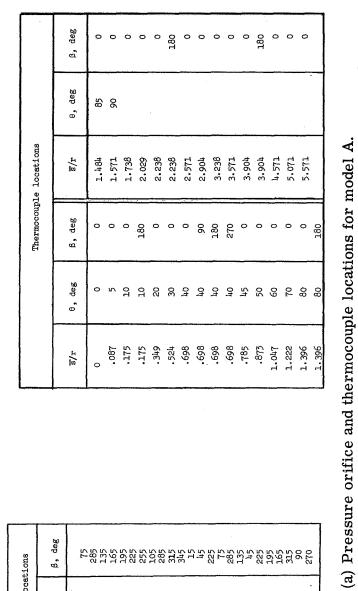
TABLE III.- HEATING DISTRIBUTIONS \dot{q}/\dot{q}_{s} on model a

θ,	β,		ġ/ġ _s fo:	r run —
deg	β, deg	5/1	14	15
0	0	0	1.000	1.000
5	0	.087	.990	.950
10	0	.175	.961	.888
10	180	.175	.907	.848
20	· 0	.349	.885	.826
30	0	.524	.740	.720
30	180	.524	.871	.807
40	0	.698	.564	.557
50	0	.873	.475	.472
60	0	1.047	.356	.345
60	90	1.047	.362	.357
60	180	1.047	.368	.353
60	270	1.047	.377	.367
70	0	1.222	.246	.237
80	0	1.396	.1343	.1398
85	0	1.484	.0888	.0881
90	180	1.571	.0938	.0889
	0	1,713	.0837	.0799
	180	1.713	.0916	.0874
	0	1.855	.0740	.0721
	0	2.140	.0679	.0687
	180	2.140	.0673	.0686
	0	2.424	.0563	.0611
	0	2.709	.0590	.0575
	180	2.709	.0690	.0698
	0	3.278	.0499	.0521

TABLE IV.- HEATING DISTRIBUTIONS $~\dot{q} / \dot{q}_{\rm S}~$ On model C







orifice locations	ß, deg	245 245 245 255 255 255 255 255 255 255
	θ, deg	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~
Pressure o	<u>s</u> /r	0.087 .175 .3175 .524 .524 .524 .578 .785 .7785 .7785 .7785 .7785 .7785 .7782 .57711 .577111 .57711 .57711 .57711 .5771111111111

Figure 1. - Instrumentation location and photograph of model A.

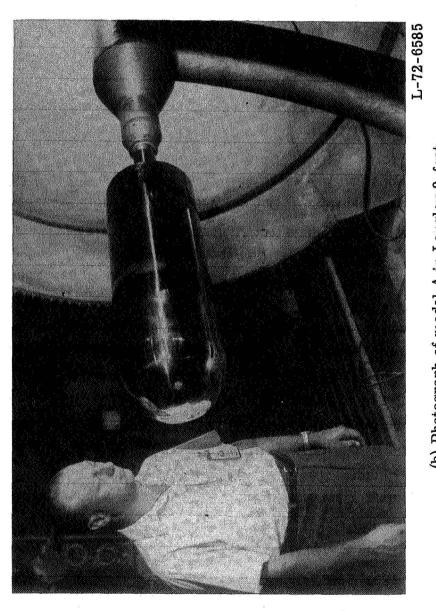
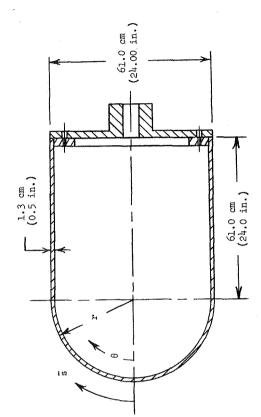


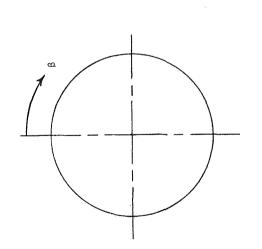
Figure 1.- Concluded.

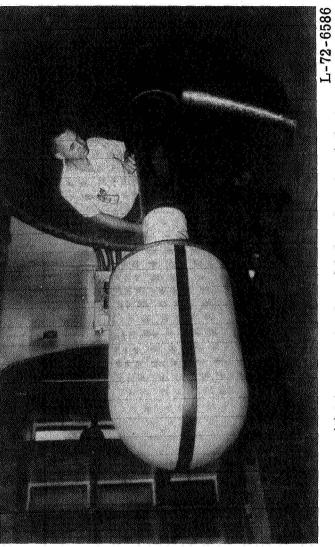
(b) Photograph of model A in Langley 8-foot high-temperature structures tunnel.

(a) Sketch and pressure orifice locations for model B. Figure 2.- Pressure orifice locations and photograph of model B.

tions	ß ,deg	Ō	0	06	180	270	0	180	0	180	0	180
Pressure orifice locations	9, deg	0	30	30	30	30	60	60				
	<u>s/r</u>	0	.524	.524	. 524	.524	1.047	1.047	170.5	2.071	3.071	3.071







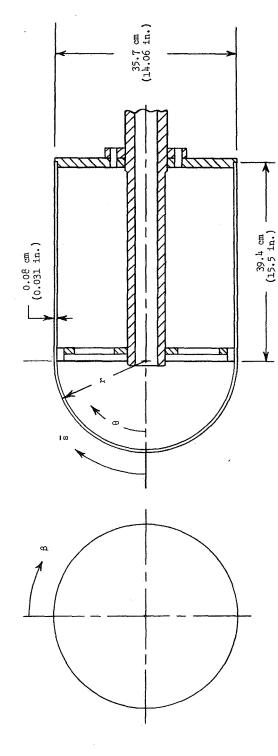
(b) Photograph of model B in Langley 8-foot high-temperature structures tunnel.

Figure 2. - Concluded.

Figure 3.- Sketch, table of thermocouple locations, and photograph of model C.

				- -			integ						<u>.</u>	<u></u>	
Thermocouple locations	gəb (f	0	0	0	180	0	180	0	Ö	7 081	0	ò	180	0	
	θ, deg	20	80	85	06		,								
	<u>s</u> /r	1.222	1.396	1.484	1.571	1.713	1.713	1.855	2.140	2.140	2.424	2.709	2.709	3.278	
	β, ủeg	0	0	0	180	0	0	180	0	0	.0	96	180	270	
	θ, đeg	0	5	IO	ТO	20	30	30	40	50	60	60	60	60	
	s/r	0	.087	.175	.175	.349	.524	.524	.698	.873	1.047	T.047	1.047	1.047	





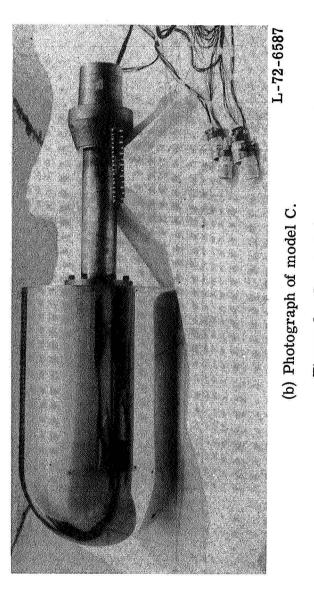


Figure 3.- Concluded.

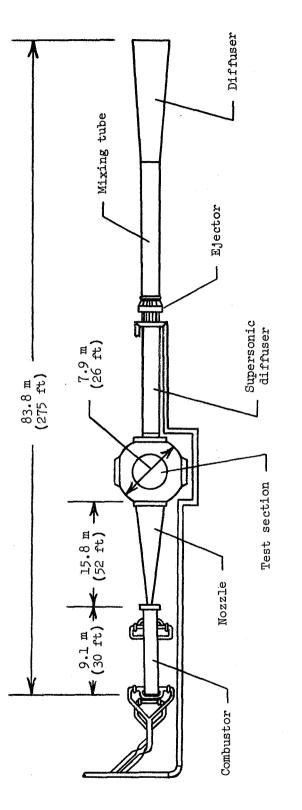




Figure 5.- Longitudinal nondimensional pressure distributions on model A at $\alpha = 0^{\circ}$.

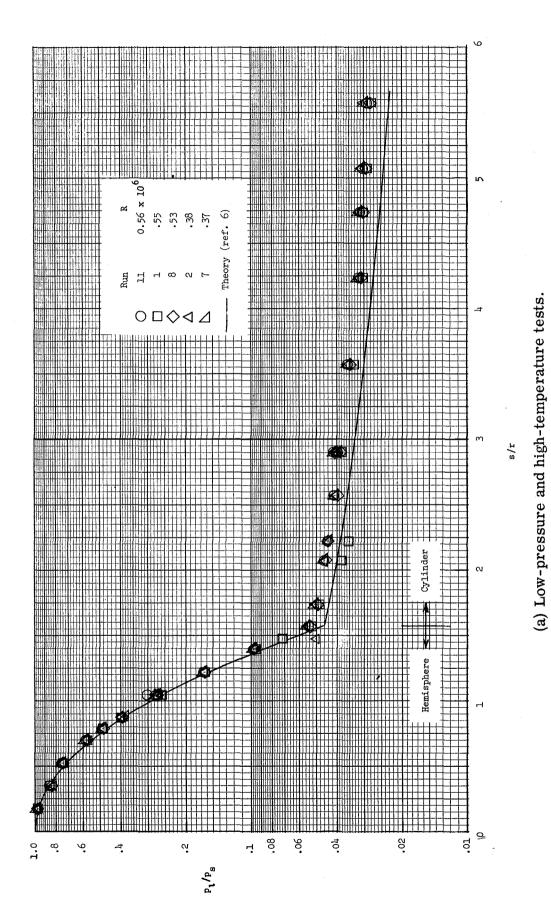
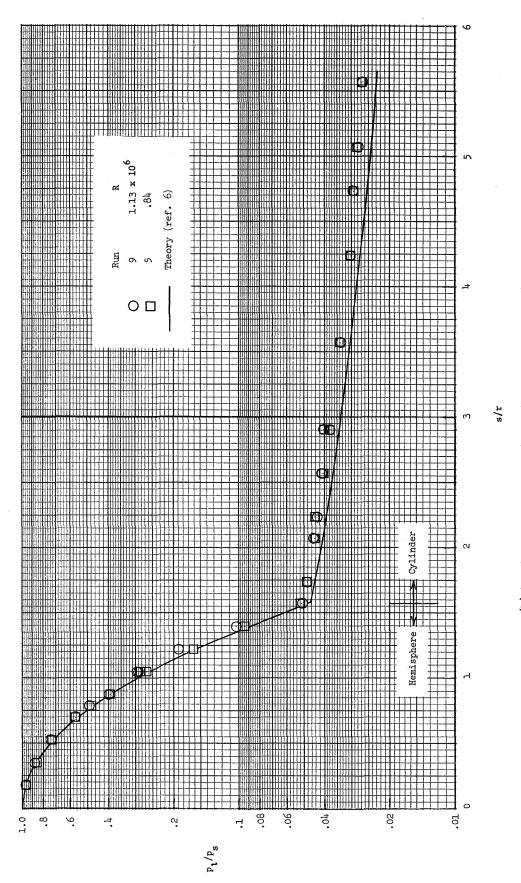


Figure 5. - Continued.

(b) High-pressure and high-temperature tests.



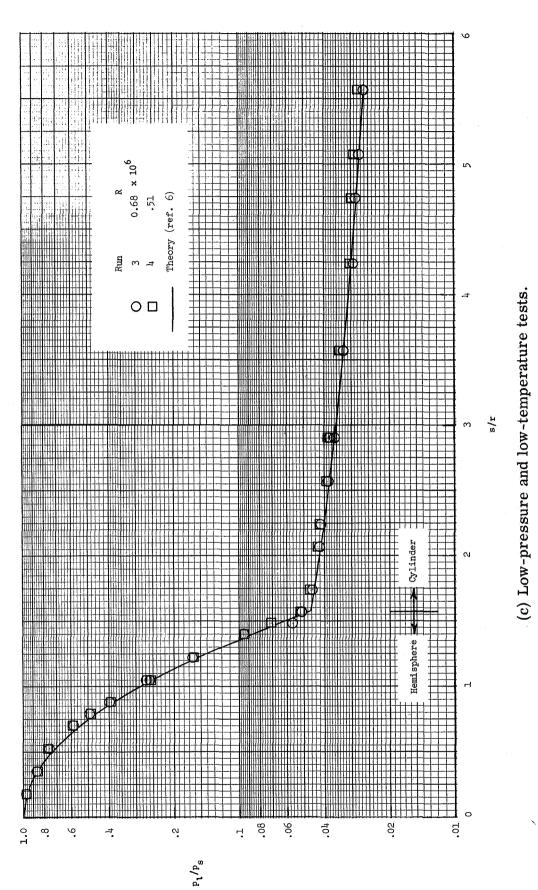


Figure 5. - Continued.

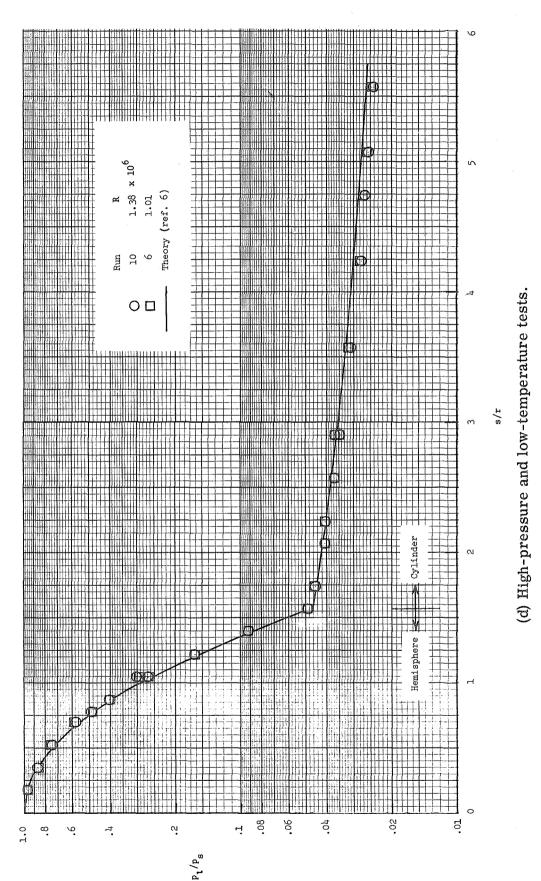


Figure 5.- Concluded.

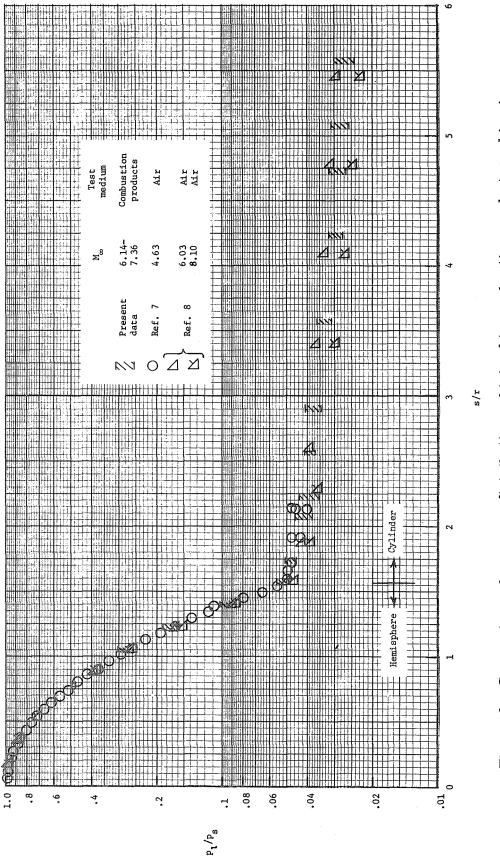
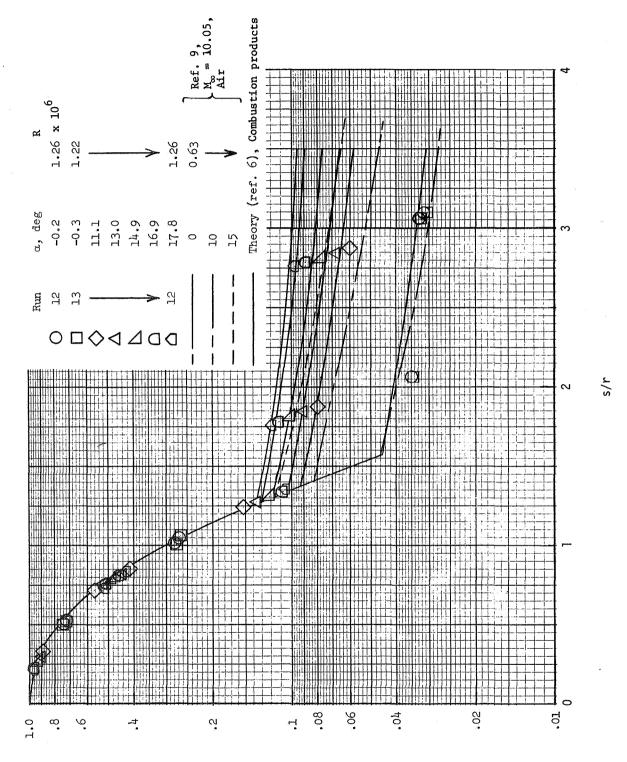
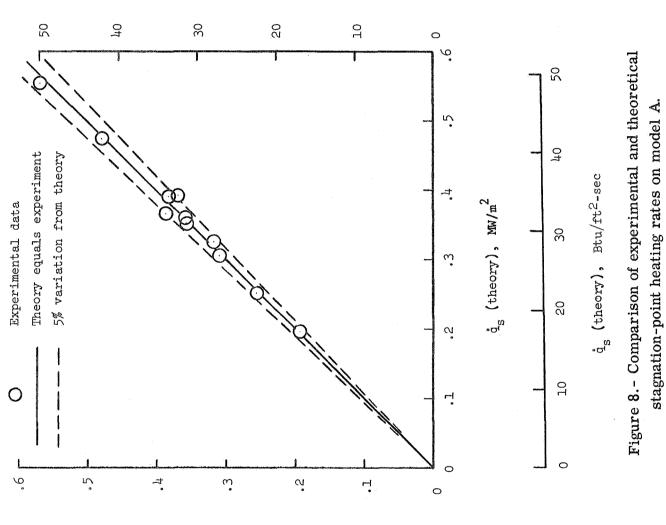


Figure 6. - Comparison of pressure distributions obtained in combustion products and in air.



p1/p





 $^{2}_{m}\mbox{WM}$,(Lstnemireqre) $_{\rm z}\dot{p}$

is (experimental), Btu/ft²-sec

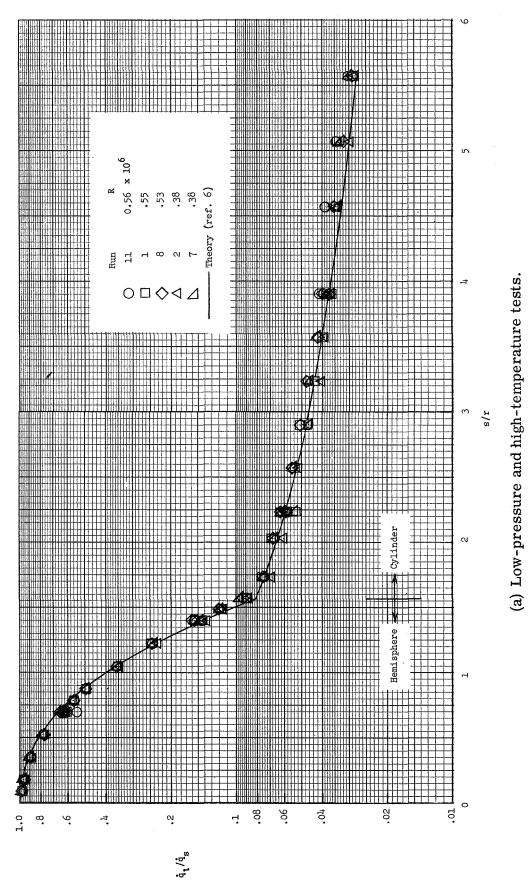


Figure 9.- Longitudinal nondimensional heating distributions on model A at $\alpha = 0^{\circ}$.

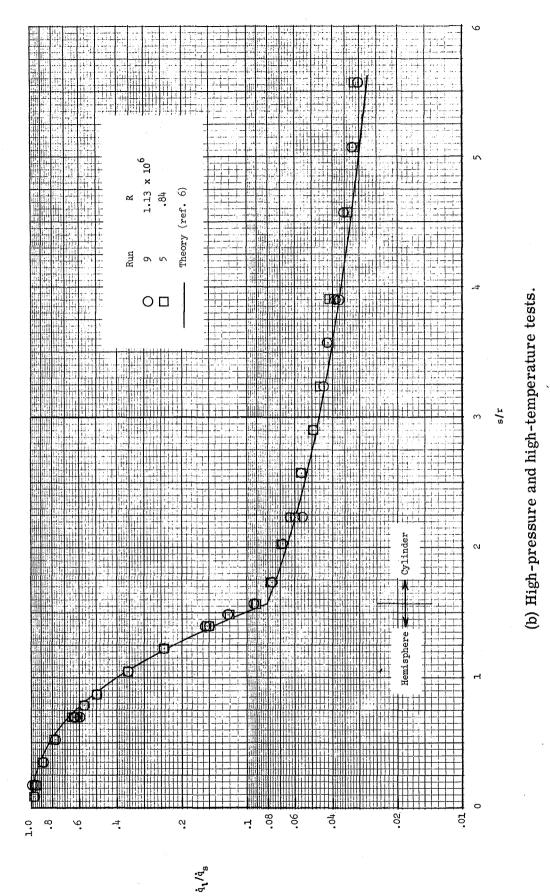


Figure 9. - Continued.

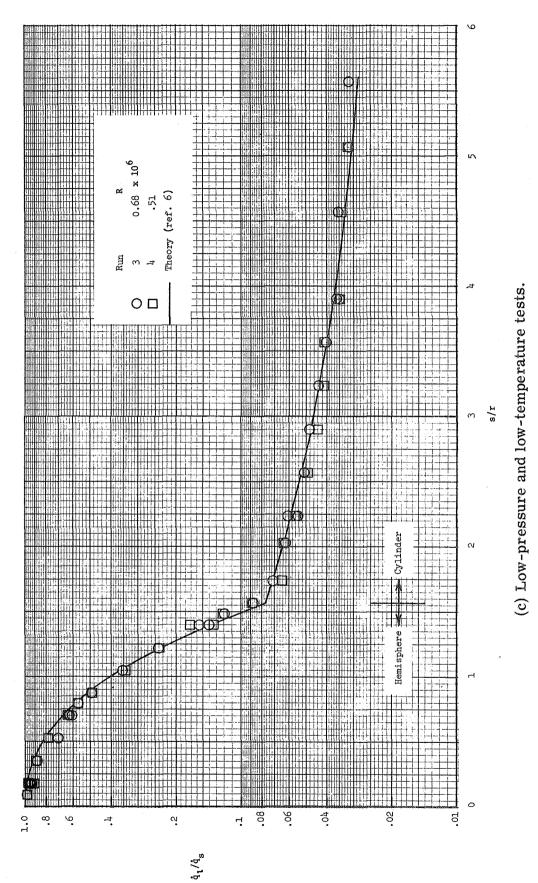
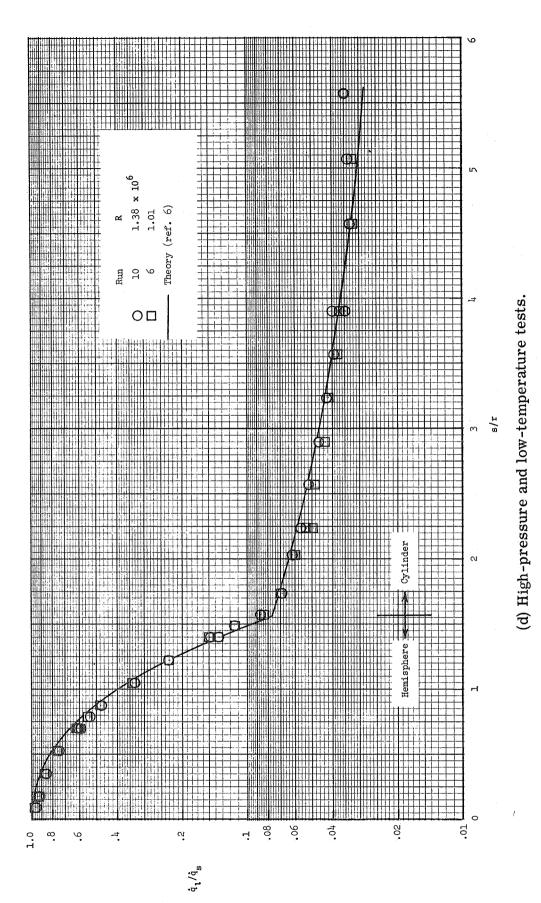
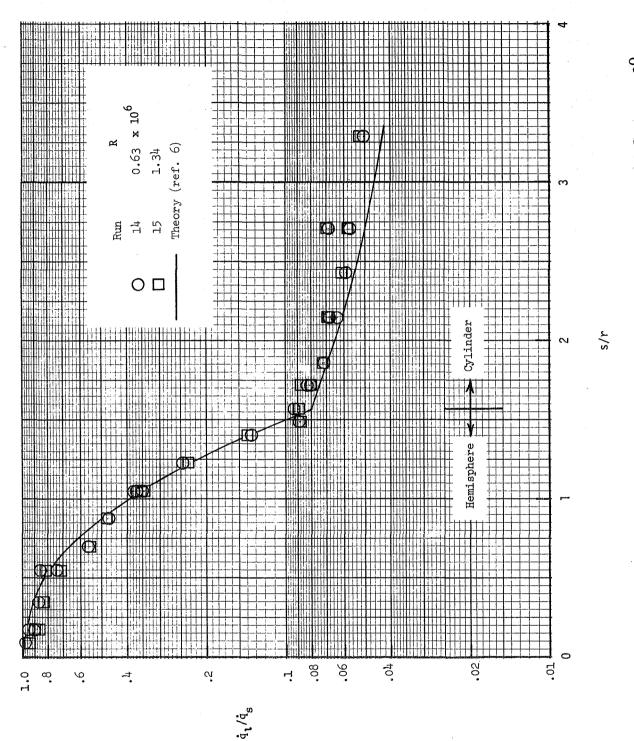


Figure 9. - Continued.

Figure 9. - Concluded.





 $\alpha = 0^{0}$. Figure 10.- Longitudinal nondimensional heating distributions on model C at

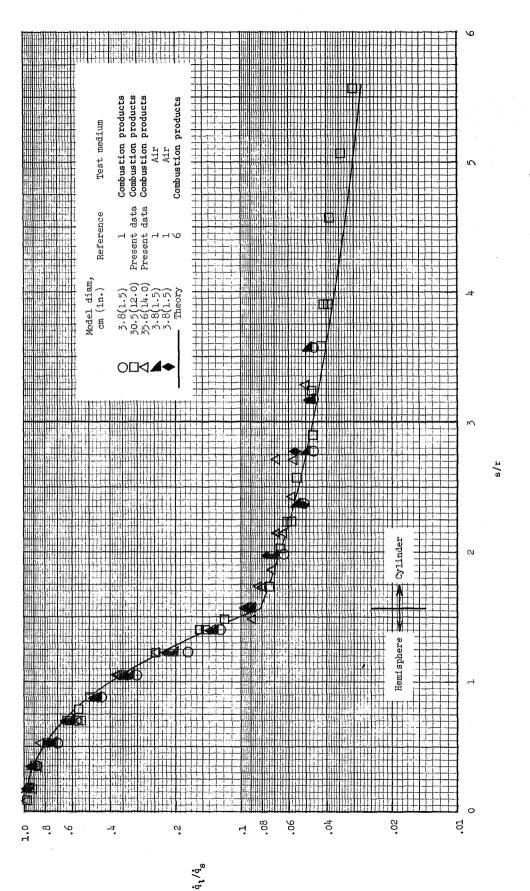
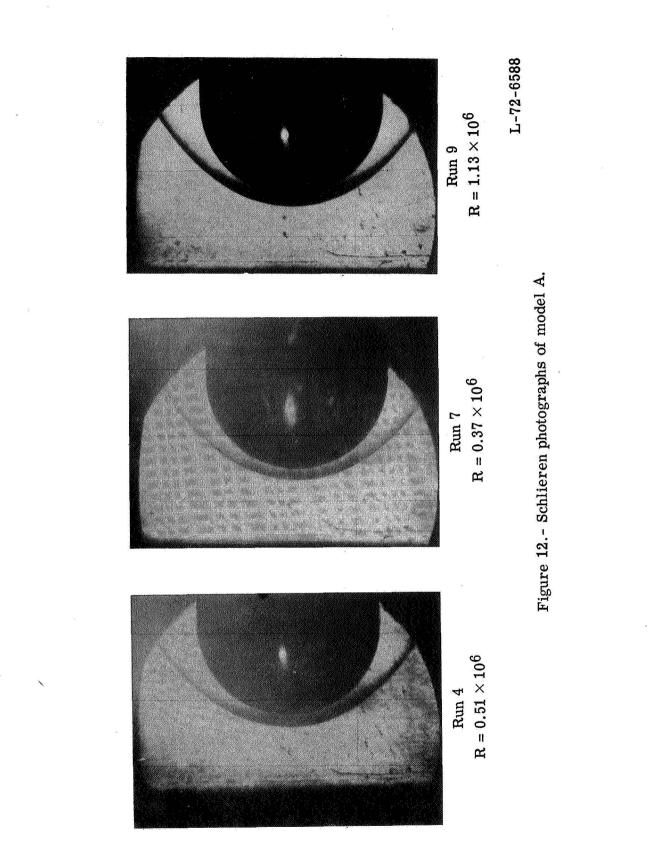


Figure 11.- Correlation of heating distributions obtained in combustion products and in air.



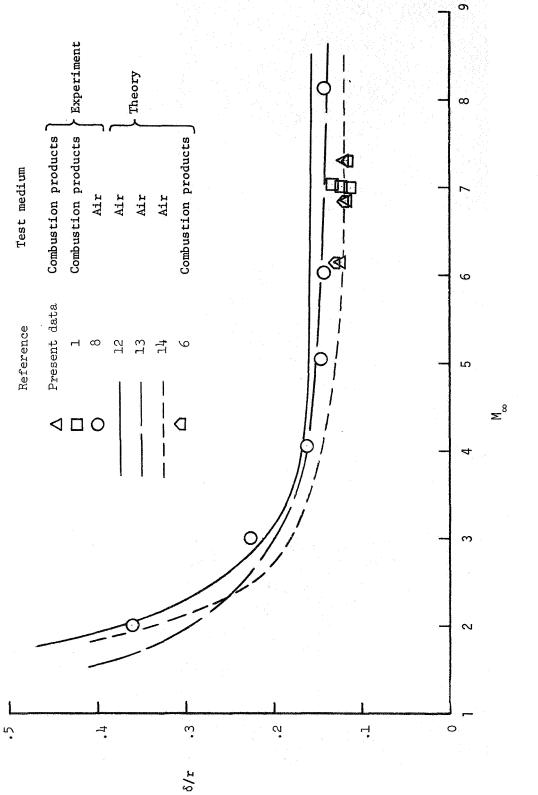


Figure 13.- Experimental and theoretical shock-wave standoff distances.

NASA-Langley, 1973 ---- 33 L-8359

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